## ENERGY-BASED ESTIMATION OF DAMPING PARAMETERS IN MULTI-DEGREE-OF-FREEDOM VIBRATION SYSTEMS

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## Introduction

This work focuses on balancing applied and dissipated energy to estimate parameters [1,2], in this case for multi-degreeof-freedom (MDOF) and large-order systems.

## Energy Balancing for MDOF Systems: An Example

The equaion of motion of a chain of four unit masses connected by unit linear springs, with linear and nonlinear damping, is  $\hat{\mathbf{M}}\ddot{\mathbf{x}} + \hat{\mathbf{C}}\dot{\mathbf{x}} + \hat{\mathbf{K}}\mathbf{x} + \hat{\mathbf{f}}(\dot{\mathbf{x}}, \mathbf{x}) = \hat{\mathbf{r}}(t)$ , where *x* is the displacement vector and  $\hat{\mathbf{r}}(t)$  is the input vector, with proportional linear damping, such that  $\hat{\mathbf{C}} = c\hat{\mathbf{K}}$ , *c* unknown, and the uniform quadratic damping  $d\dot{\mathbf{x}}_i | \dot{\mathbf{x}}_i |$  with unknown constant *d* is applied to each mass. The excitation  $a \cos \omega t$  is applied to the fourth mass. We simulated responses with the values c = 0.15 and d = 0.10, at a step size of h = 0.1 for 1000 steps to remove the transients, and then 200 more steps to record steady state responses. We resonated the first mode ( $\omega = 0.618$ ) with input amplitudes a = 0.25, a = 0.5, a = 0.75, and a = 1. Figure 1 shows the animated response through a cycle of vibration for a = 1.0.

The positions are measured during a periodic response. The identification equations are formed by the inner product of the velocity vector with the equation of motion. Integrating over a period of oscillation, the conservative terms drop out, and the nonconservative work terms balance as

$$W_d = \oint \dot{\mathbf{x}}^T \hat{\mathbf{C}} \dot{\mathbf{x}} dt + \oint \dot{\mathbf{x}}^T \hat{\mathbf{f}}(\dot{\mathbf{x}}) dt = \oint \dot{\mathbf{x}}^T \hat{\mathbf{r}}(t) dt = W_a.$$
(1)

Equation (1) has embedded unknowns *c* and *d*, and has the form  $c\alpha + d\beta = \gamma$ , where  $\alpha = \oint (\dot{x}_1^2 + \dot{x}_2^2 + \dot{x}_3^2 + \dot{x}_4^2 - \dot{x}_1\dot{x}_2 - \dot{x}_2\dot{x}_3 - \dot{x}_3\dot{x}_4)dt$ ,  $\beta = \oint (|\dot{x}_1|^3 + |\dot{x}_2|^3 + |\dot{x}_3|^3 + |\dot{x}_4|^3)dt$ , and  $\gamma = \oint \dot{a}x_4 \cos(\omega t)dt$ . These integrals were performed using the rectangular integration rule. For each excitation amplitude, we completed this equation, and the least squares solution led to estimates  $c_I = 0.1499$  and  $d_I = 0.1003$ .

If there are more degrees of freedom than sensors, we use proper orthogonal decomposition (POD) for reduced-order modeling. Suppose the "dominant" proper orthogonal modes (POMs) are determined (from the eigenvectors of  $\mathbf{R} = \mathbf{X}^T \mathbf{X}/N$ , where the rows of  $\mathbf{X}$  are the *N* time samples of elements of  $\mathbf{x}$ ). We write  $\mathbf{x} \approx \mathbf{U}\mathbf{y}$ , where columns of  $\mathbf{U}$  are the dominant POMs, and  $\mathbf{y}$ are the proper orthogonal modal coordinates. If retained modes represent a large percentage of the signal power (for example 99.9%), the approximate equality is rather good. Substituting into the equation of motion and premultiplying by  $\mathbf{U}^T$  yields

$$\mathbf{U}^{T}\hat{\mathbf{M}}\mathbf{U}\ddot{\mathbf{y}} + \mathbf{U}^{T}\hat{\mathbf{C}}\mathbf{U}\dot{\mathbf{y}} + \mathbf{U}^{T}\mathbf{K}\hat{\mathbf{U}}\mathbf{y} + \mathbf{U}^{T}\hat{\mathbf{f}}(\mathbf{U}\mathbf{y},\mathbf{U}\dot{\mathbf{y}}) \approx \mathbf{U}^{T}\hat{\mathbf{r}}(t), \quad (2)$$

or  $\mathbf{M}\ddot{\mathbf{y}} + \mathbf{C}\dot{\mathbf{y}} + \mathbf{K}\mathbf{y} + \mathbf{f}(\mathbf{y}, \dot{\mathbf{y}}) \approx \mathbf{r}(t)$ . Thus, for the energy balance step,  $\oint \dot{\mathbf{y}}^T (\mathbf{C}\dot{\mathbf{y}} + \mathbf{f}(\mathbf{y}, \dot{\mathbf{y}}))dt \approx \oint \dot{\mathbf{y}}^T \mathbf{r}(t)dt$ , which is the identification equation of the form  $c\alpha + d\beta = \gamma$ .

Using the same data, the dominant POM of each response contained 96.9%, 97.5%, 98.1%, and 98.8%, respectively, of the



Figure 1. Axial steady-state displacement plotted transversally (a = 1).

total signal energy. The identification results were  $c_I = 0.1404$  and  $d_I = 0.1138$ , the error induced by reduced order modeling.

We applied this to a string, robustly to added sensor noise. Acknowledgements: This work is related to projects with NASA (NAG-1-01048)and NSF (CMS-0099603).

## References

[1] G. R. Tomlinson and J. H. Hibbert, 1979, *Journal of Sound and Vibration* **64** 233-242.

[2] J.-W. Liang and B. F. Feeny, 2006, *Journal of Sound and Vibration* **295** 988-998.